

# Effect of Blade Torsion on Modeling Results for the Small Wind Research Turbine (SWRT)

## Preprint

D. Corbus  
*National Renewable Energy Laboratory*

A.C. Hansen and J. Minnema  
*Windward Engineering LLC*

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# Effect of Blade Torsion on Modeling Results for the Small Wind Research Turbine (SWRT)

Dave Corbus  
National Wind Technology Center  
National Renewable Energy Laboratory  
1617 Cole Boulevard  
Golden, Colorado 80401-3393 (USA)

A. Craig Hansen, Jeff Minnema  
Windward Engineering LLC  
2225 East Murray Holladay Road #201A  
Salt Lake City, Utah 84117 (USA)

**The Small Wind Research Turbine (SWRT) testing project was initiated with the goal of better characterizing both small wind turbine loads and dynamic behavior. The main purpose of the testing was to produce high quality data sets for model development and validation and to help the wind industry further their understanding of small wind turbine behavior, including furling. Previous papers have discussed modeling results for the SWRT using the FAST aeroelastic simulator. This paper summarizes modeling results from both the FAST and ADAMS aeroelastic simulators. The ADAMS model allows for blade torsional stiffness to be modeled, which is shown to significantly improve the agreement between model and test results for rotor speed and explain variances in previous comparisons between the SWRT FAST modeling results and the SWRT test data. The ADAMS model results also show that the aerodynamic pitching moment impacts the blade torsion more than the inertial forces.**

## I. Introduction

Small wind turbines provide unique challenges to researchers trying to model their performance because they have higher RPMs than large wind turbines and often operate with large yaw errors and high yaw rates. Furthermore, many small wind turbines incorporate furling, whereby the rotor either tilts and/or yaws out of the wind to protect itself from overspeed during very high winds, and this can add further complications to modeling their performance. Because of these operating conditions, modeling the performance of small wind turbines is particularly challenging.

The Small Wind Research Turbine (SWRT) project was initiated with the goal of providing reliable data for a small furling wind turbine for model development, furthering the understanding of small wind turbine aerodynamics and furling, and model validation. The test was the most extensive loads test ever conducted for a small wind turbine<sup>1</sup>, and included measurements of turbine furling and thrust, as well as many other loads measurements.

The SWRT was modeled using the FAST<sup>2,3</sup> (Fatigue, Aerodynamics, Structures, and Turbulence) code with the AeroDyn aerodynamic subroutine package developed by Windward Engineering LLC<sup>4</sup> to calculate aerodynamic forces. FAST, which was developed and is distributed by the National Renewable Energy Laboratory's (NREL's) National Wind Technology Center (NWTC), is the primary aeroelastic simulator used by the U.S. wind industry to model small wind turbines. The recent upgrades to FAST allow users to simulate lateral offset and skew angle of the rotor shaft from the yaw axis, rotor-furling and tail-furling degrees-of-freedom (DOFs), up- and down-furl stops, and tail inertia and tail fin aerodynamic effects<sup>5</sup>. Because the location and orientation of the furling DOFs are user-specified, the simulator is flexible enough to model virtually any furling wind turbine configuration. FAST's built-in features allow for the determination of full-system modes, including furling, of an operating or stationary turbine.

Extensive modeling of the SWRT turbine, using FAST, showed that rotor RPM was always over predicted, whereas the predicted and measured thrust were well correlated. This error in the rotor RPM was found for different airfoils and increased with wind speed <sup>6</sup>. Various approaches were tried to understand why the FAST model predicted rotor speeds much higher than measured in high winds. In low winds, the predictions were quite good. Both the airfoil tables and the alternator torque-speed curve were adjusted in various ways. Unrealistically large changes were required to improve the agreement in high winds, but then the accuracy in low winds was compromised. The reason for the discrepancy was finally discovered to be the elastic twist of the blade, which increases with rotor speed (or wind speed).

Subsequent modeling with ADAMS<sup>®</sup> (Automatic Dynamic Analysis of Mechanical Systems, “ADAMS” is used to imply “ADAMS<sup>®</sup>” throughout this paper) <sup>7</sup>, allowed blade torsion to be modeled. ADAMS is a general purpose, multibody-dynamics code from the MSC Software Corporation for calculating multi-body dynamic forces that is adaptable for modeling wind turbines. FAST cannot currently model blade torsion.

The approach to modeling the SWRT consisted of the following: (1) upgrading the FAST model to include furling; (2) validating the FAST model by constructing an ADAMS model of the SWRT and comparing the two results for furling cases; (3) running simulations with the FAST model and comparing them to the SWRT test data; and finally, (4) including torsional blade stiffness in the ADAMS model and re-running the simulations performed in step (3) in ADAMS. This last step, which is the topic of this paper, was necessary because after lengthy data analysis and comparisons between FAST SWRT simulations and test data, it was determined that the “live twist” of the blade was having a significant impact on rotor performance. Torsional stiffness cannot currently be modeled in FAST, so the ADAMS model was used for the final simulations and the torsional stiffness of the blade was input into the model.

The ADAMS modeling with blade torsion as an input resulted in a significantly improved correlation between modeled and measured rotor RPM as discussed in this paper and revealed that blade torsion, referred to herein as “live twist”, can have a very significant effect in wind turbine modeling.

## II. Test Turbine Description

The SWRT is a modified Bergey EXCEL 10-kW turbine that furls horizontally out of the wind. The tail boom attaches with a hinge and bumper arrangement to the rotor/generator/main-frame assembly, and there is a lateral offset between the yaw axis and the rotor axis. The axis of the hinge pivot joint is inclined laterally at a small angle to the vertical yaw axis and produces a gravity restoring moment. A main-frame stop keeps the tail boom from furling more than about 68 degrees. As the wind speed increases, so do the thrust and aerodynamic normal force on the nominally aligned vertical tail at the end of the tail boom. Furling occurs when the rotor moments exceed the gravity restoring moment. The furl damper provides very little resistance to furling but a high resistance to unfurling.

The Bergey EXCEL is a three-bladed upwind turbine with a rated output of 10 kW at 13.0 m/s. The EXCEL uses a permanent magnet alternator to produce three-phase variable frequency output at a nominal 240-volts. The three-phase output is then rectified to DC power and converted to single-phase 240-volt 60-hz AC power by the Gridtek inverter. The turbine blades are constant chord and made from pultruded fiberglass, and the direction of rotation is counter-clockwise. The experimental data in this paper are from Configuration C of the SWRT, which has constant chord blades with the SH3055 airfoil, a swept area of 35.3 m<sup>2</sup>, and a rotor diameter of 6.7 m. The turbine nacelle and shaft have an eight-degree tilt, and the SWRT is installed on a Rohn SSV, 24.4 m. (80 ft.), freestanding lattice tower.

The SWRT was modified in several ways to allow for the installation of test instrumentation. One significant modification was the turbine main shaft was shortened approximately 0.18 m. (7 in.) to allow for installation of a 0.18 m (7 in.) load fixture in-line with the non-rotating shaft to measure shaft loads. It should be noted that because the SWRT was modified, the test results in this report are not necessarily representative of the Bergey EXCEL turbine.

## III. Modeling Approach

To supply inputs for aeroelastic models of the SWRT, the turbine tail assembly and mainframe were weighed and centers of gravity (Cgs) were determined. Tests were also conducted to calculate the inertia about the tail axis and the yaw axis. Tail damper properties were measured and all turbine geometries noted. A modal test was conducted for a blade to determine mode shapes for flap and edge. This data, as well as all turbine related parameters, including airfoil data, was input into the FAST and ADAMS models of the SWRT. To measure the blade torsion, 182.3 N-m of torque was applied about the 0.25 chord of the blade. This was applied 2.74 m from the

outboard edge of the rootpad (twisting 2.74 m of blade) and the measured angular deflection was 4.1 degrees or .022 degrees/N-m over 2.74 m of blade. Based on this data the torsional stiffness of the SH3055 blade is 6500 Nm. The c.g. offset is 0.017 m (towards the trailing edge) from the quarter chord.

The FAST and ADAMS models of the SWRT were developed using geometric, aerodynamic, and mass properties for the SWRT. The models include blade flexibility, a variable-speed generator with a torque-speed look-up table, tail furling with nonlinear user-defined springs and dampers, and free yaw.

Modeling data was compared to experimental data taken for the SWRT turbine <sup>1</sup>. Data were sampled at 160 hz, and a 40-hz, 6-pole low-pass Butterworth filter was used to filter the data. Records of 10 minutes in length were stored.

#### IV. SWRT Modeling

##### A. Blade Torsion

Because the FAST program cannot model blade elastic torsion motion, an ADAMS model was created from the FAST model that includes the blade torsion degrees of freedom (DOF). The torsional stiffness of the blades was measured and input into the ADAMS model. The aerodynamic pitching moment was added to the airfoil tables so the ADAMS model includes both aerodynamic and inertial causes of blade twisting. There were no other changes to the model. Therefore, the only substantial reason for differences between the FAST and ADAMS predictions is live blade twist. (Previous work has shown that FAST and ADAMS predictions are nearly identical when blade torsion stiffness is very high <sup>5</sup>).

ADAMS and Fast predict blade motions differently. ADAMS uses 6 DOF per blade element, and there are 15 elements per blade for a total of 90 DOF per blade. Fast has 3 DOF per blade ( 2 flapwise and 1 edgewise) plus a polynomial mode shape for each DOF. ADAMS also requires blade torsional stiffness values, chordwise center of gravity (Cg) offsets, and the option of using the center of mass (Cm) values in the airfoil tables. Fast does not allow for any of these.

##### B. Modeling Results for Summary Statistics

Direct comparison of time series data is useful in that it provides a highly detailed view of a small amount of data. Comparisons of 10-minute average, maximum, and minimum summary statistics are valuable because they provide a view of a large quantity of data, albeit in much less detail. The statistics were compared by running the ADAMS model with simulated turbulent wind conditions generated by SNWind v1.22. An 8 x 8 grid of points was used across the rotor disc and three components of time-varying wind were generated at each grid point. The turbulence was generated with statistics that match the IEC Category-A conditions. Site-specific turbulent inputs were not used in this analysis. Figure 1 shows a plot of the actual turbulence intensity from the SWRT tests and the IEC Category A turbulence.

Figures 2-7 show the results of the summary statistics for SWRT Configuration C using the ADAMS model. These results were generated using the original ‘untuned’ airfoil and alternator data. They therefore represent the accuracy of predictions that would be done during a design if airfoil and alternator properties were available from wind tunnel or bench testing and blade elastic properties were known from Finite Element Analysis (FEA) or testing.

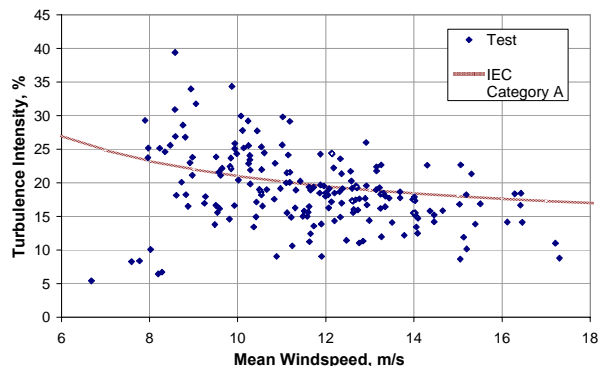


Figure 1. IEC Category A and test turbine turbulence intensities

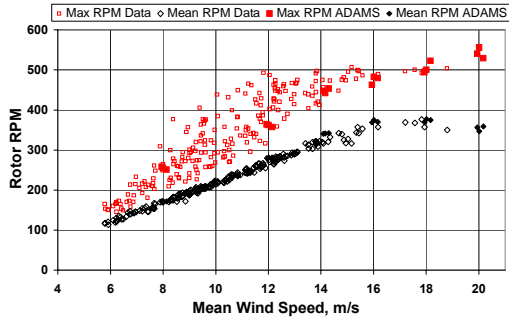


Figure 2. 10-Minute mean and maximum values of rotor RPM for Configuration C.

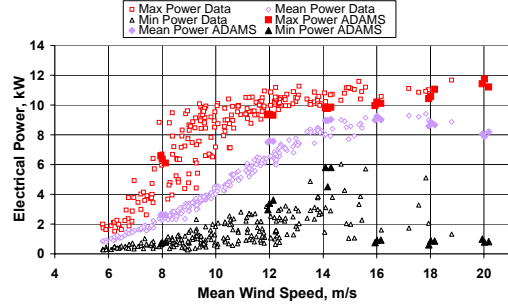


Figure 5. 10-Minute mean, maximum, and minimum values of electric power for Configuration C.

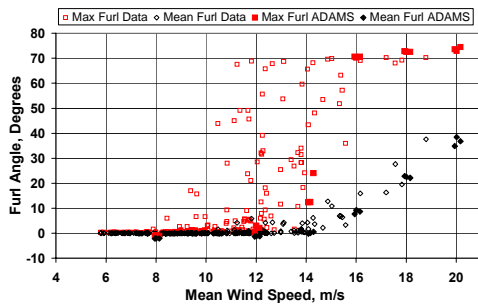


Figure 3. 10-Minute mean and maximum values of tail furl for Configuration C.

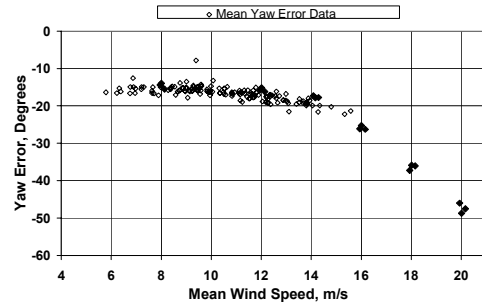


Figure 6. 10-Minute mean values of yaw error for Configuration C.

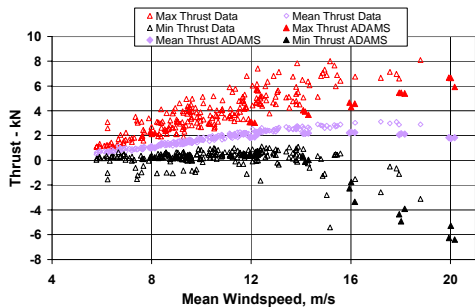


Figure 4. 10-Minute mean, minimum, and maximum values of thrust for Configuration C.

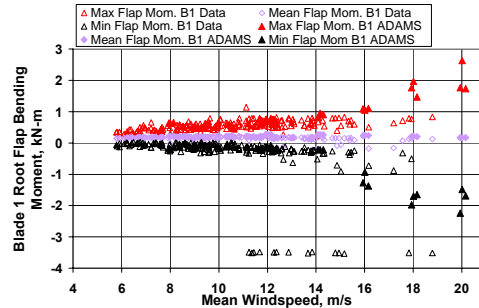


Figure 7. 10-Minute mean, maximum, and minimum values of Blade 1 flap bending for Configuration C.



### C. Time Series Modeling Results

Time series comparisons are made by driving the ADAMS model with measured wind data. Measurements of the hub-height horizontal and vertical wind speeds and the wind direction are used for the comparisons. Measurements of instantaneous horizontal wind shear were not available. Horizontal wind shear is known to be an important driver of yaw and furling motions<sup>8</sup>, but it is difficult to measure. Driving a model with “hub-height” wind data misses small-scale turbulence effects but captures the large-scale, slow variations in wind characteristics.

Figure 8 is a plot of blade elastic twist versus rotor RPM. The twist can be up to 4 deg. at higher RPMs. Figure 9 is a time series plot of blade elastic twist for the SWRT that shows the large amount of live twist during one 10-minute time series file with an average wind speed of 16.2 m/s. Figures 10-13 show the same time series file for RPM, furl, thrust, and yaw error. The FAST modeling results without torsional stiffness are also shown in addition to the ADAMS results with torsional stiffness. The agreement between the ADAMS model data and test data is good, whereas the FAST model severely over predicts rotor RPM, and furl and yaw error agreement are also poor. The results demonstrate the importance of blade live twist to modeling, especially for the RPM.

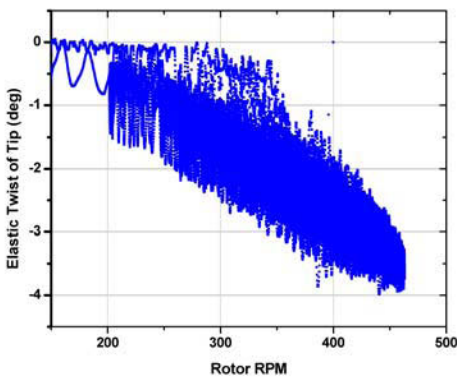


Figure 8. Elastic twist versus rotor RPM for one 10-minute test of Configuration C.

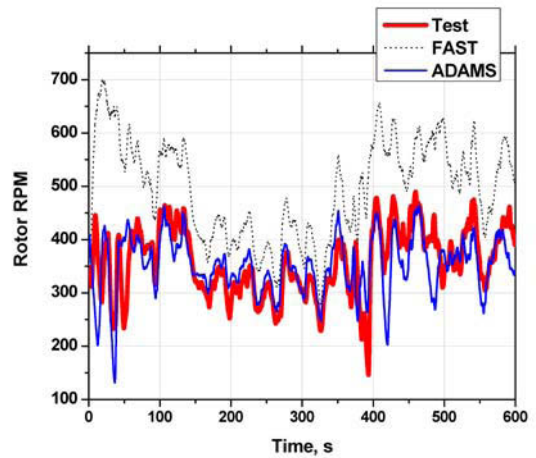


Figure 10. Rotor RPM versus time for same 10-minute test.

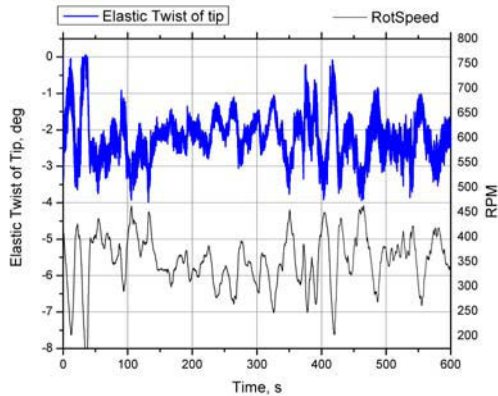


Figure 9. Elastic twist of blade tip and rotor RPM versus time for same 10-minute test (*negative twist is toward feather*).

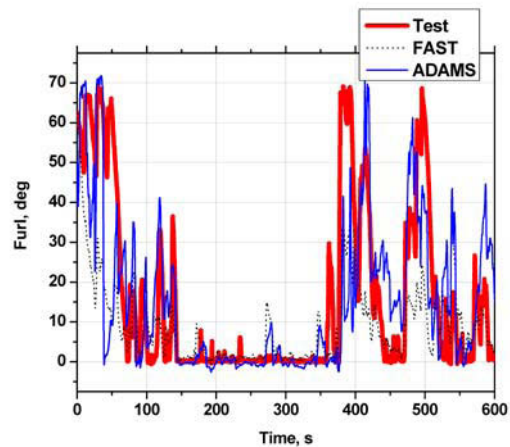
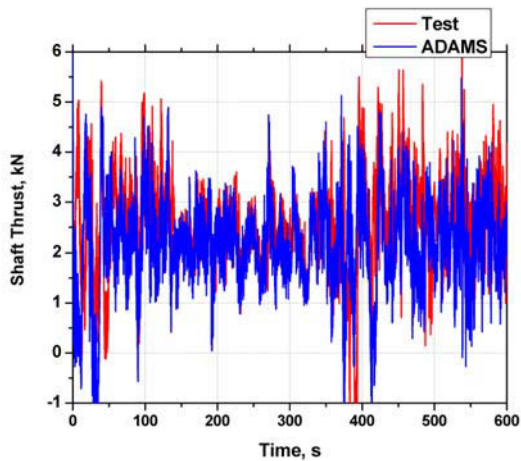
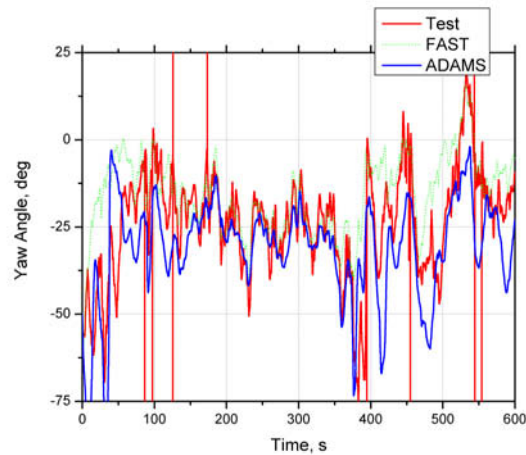


Figure 11. Rotor furl versus time for same 10-minute test.



**Figure 12. Rotor thrust versus time for same 10-minute test.**



**Figure 13. Yaw error versus time for same 10-minute test.**

#### **D. Effect of Aerodynamic Pitching Moment and Wake Model Algorithm on ADAMS Results**

Modeled sensitivity analyses with ADAMS show the importance of the aerodynamic pitching moment to the blade live twist. Figure 14 shows a plot of rotor speed versus time for the same 10-minute data series shown previously with the aerodynamic pitching moment turned both on (i.e. Cm) and off (i.e. No Cm). The data show aerodynamic forces have a large impact on “live” blade twist. Live blade twist has a big impact on RPM and hence power. Figure 15 shows a plot of electric power output versus time for the same 10-minute data set.

Figures 14 and 15 also show the effect of changing the dynamic inflow option in ADAMS. When the value is DYNIN<sup>4</sup>, a generalized dynamic wake (GDW) inflow model is used to calculate the induction factor. A value of EQUIL<sup>4</sup> assumes that the wake is always in equilibrium with the forces on a blade element (the “quasi-steady” or equilibrium wake assumption). For EQUIL, the BEM theory with skewed wake and tip loss corrections is used. The GDW model was developed as an expanded version of the Pitt and Peters model. It uses a series solution to describe the induced velocity field in the rotor plane, which includes Legendre functions in the radial direction and trigonometric functions in the tangential direction. The current GDW model in AeroDyn employs 0P, 1P, 2P and 3P terms<sup>4</sup>. The dynamic inflow effect is often insignificant, so results in most cases should not differ from the BEM results. The exception is cases with rapid changes in blade angle-of-attack, where the dynamic inflow effect can be significant, as is the case with the SWRT turbine. Yaw oscillation resulting from the GDW model has also been observed while modeling a small, upwind, tail-vane turbine, though the reasons for this are not yet known<sup>4</sup>.

#### **E. Comparison of Rotor RPM between ADAMS and SWRT Data**

Comparisons between predicted rotor RPM for the ADAMS model and measured rotor RPM from the SWRT data show some interesting trends depending on if the GDW or BEM wake model is used. Figure 16 shows summary statistics for rotor RPM versus wind speed for the ADAMS model results with the two different wake models and the SWRT data for Configuration C. The aerodynamic pitching moment is turned on in these ADAMS plots.



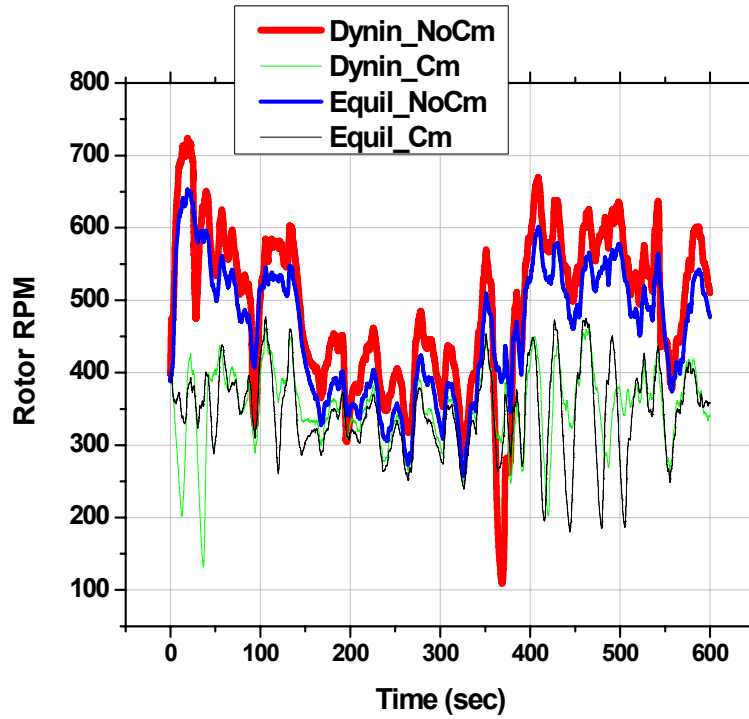


Figure 14. 10-Minute mean and maximum values of rotor RPM for Configuration C.

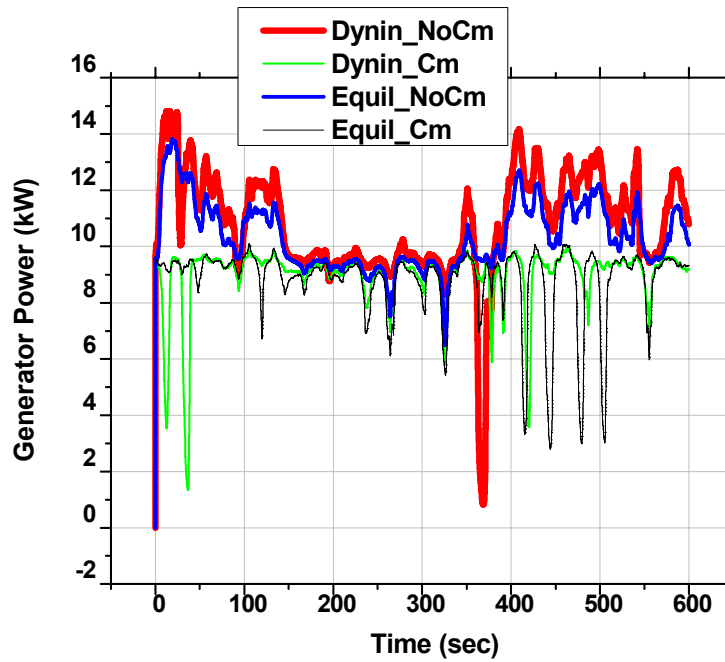
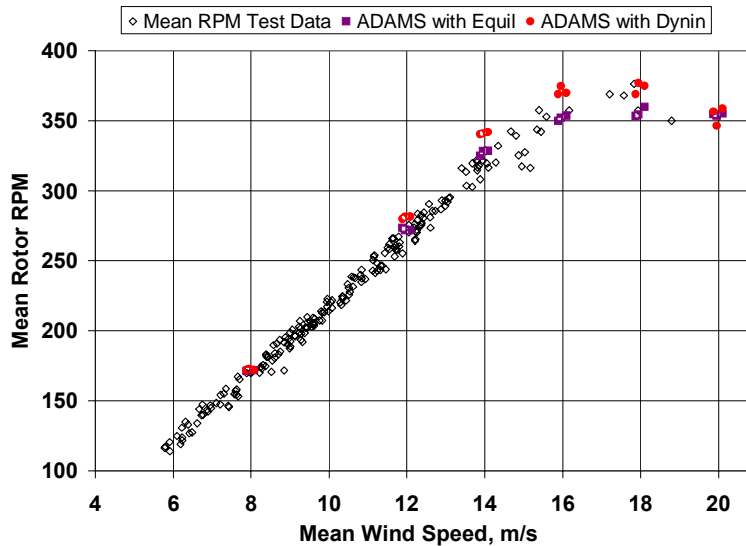


Figure 15. 10-Minute mean and maximum values of rotor RPM for Configuration C.



**Figure 16. Rotor RPM versus wind speed for ADAMS with two different wake models and SWRT data.**

## V. Conclusion

The SWRT test has provided the small wind turbine and research community with a unique set of data for validating aerodynamic models for a turbine that operates at a significant yaw error, has high yaw rates, is in and out of stall quickly, and operates at higher RPMs. Analysis and comparison of FAST and ADAMS model simulations and test data for the SWRT has resulted in a better understanding of small wind turbine modeling, as well as a FAST furling model that can be used by engineers to better design small furling wind turbines. However, comparison of FAST and ADAMS results has shown the importance of including blade torsion in the SWRT model, which can only be done with the ADAMS model at this time. The SWRT blade is quite soft in torsion; hence it is doubtful that live twist will be as important for many other turbines. It was shown with the ADAMS model that the aerodynamic pitch moment has more of an effect on blade torsion than do inertial forces. Future near-term changes to the FAST model will include adding the ability to model blade torsion.

## Acknowledgments

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